



Beverage machine Cooling Analysis

# Content

## ❖ Design requirement

- 5L water be cooled from 25°C to 4°C
- Must cool down less than 30min

## ❖ Software benchmark

- Cylinder geometry
- Fluid behavior

## ❖ Boundary conditions

- Transient analysis
- TEC
- Air/ Liquid cooling



# Design Requirement



∞ 5L water,  $T_0 = T_a = 25^\circ\text{C}$ ,  $T_{1800\text{s}} = 4^\circ\text{C}$

∞ Properties of Water

- Density:  $997\text{kg/m}^3$
- Specific heat:  $4181.6\text{J/kg K}$
- Thermal Conductivity:  $0.607\text{W/mK}$

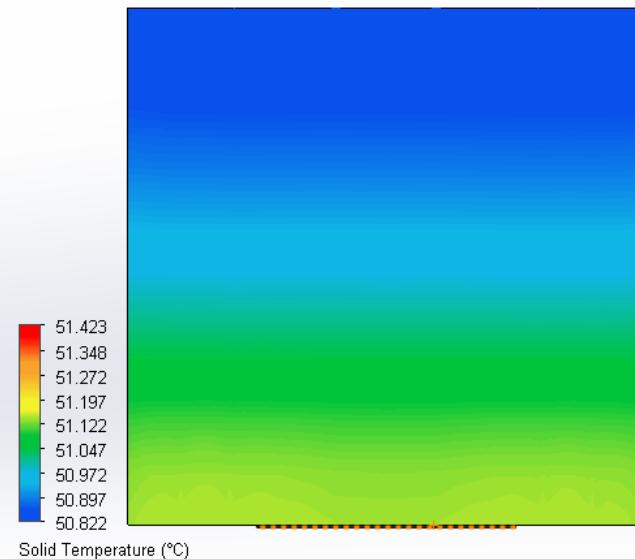
∞ Internal energy change

- $\Delta U = mC_p\Delta T = 0.005 \times 997 \times 4181.6 \times (25-4) = 437,750 \text{ J}$
- 30min to reach the target temperature
  - Average power is  $437750/1800=243.2\text{W}$
  - If the TEC cooling capability is 50W, we need at least 5

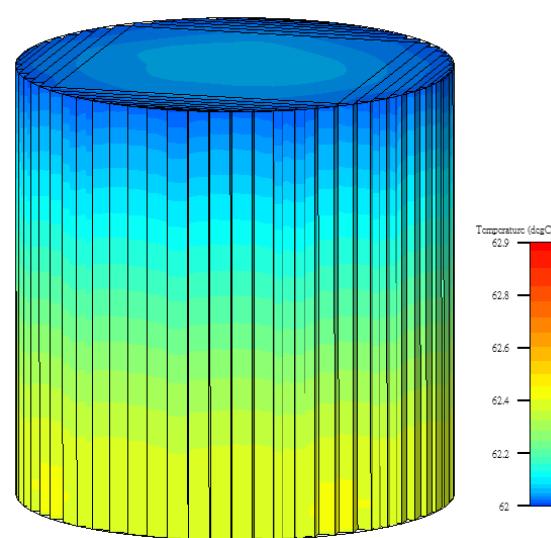
# Geometry Impact



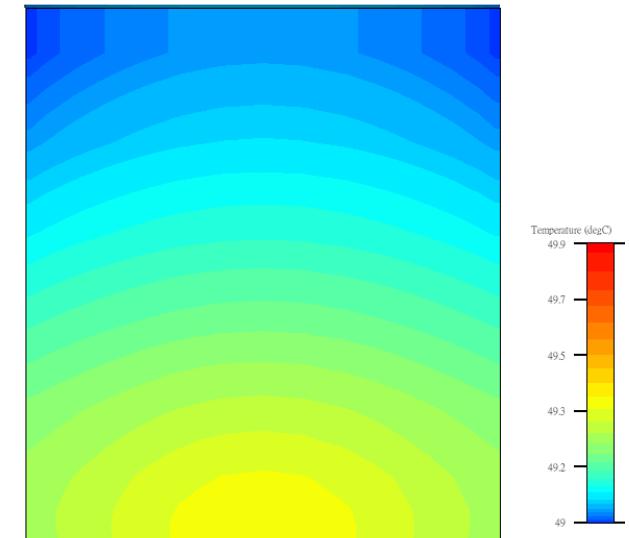
- ☞ Consider the natural convection and radiation effects
  - Simple cylinder model to check the convection
  - 60W loading, check surface temperature by solid conduction
  - $\varepsilon=1$ ,  $D=200\text{mm}$ ,  $H=200\text{mm}$ ,  $T_a=20^\circ\text{C}$ ,  $T_{\text{surface}}=53^\circ\text{C}$  by theoretical solution



FloTHERM XT Cylinder  
 $T_{\text{surface}}=51^\circ\text{C}$



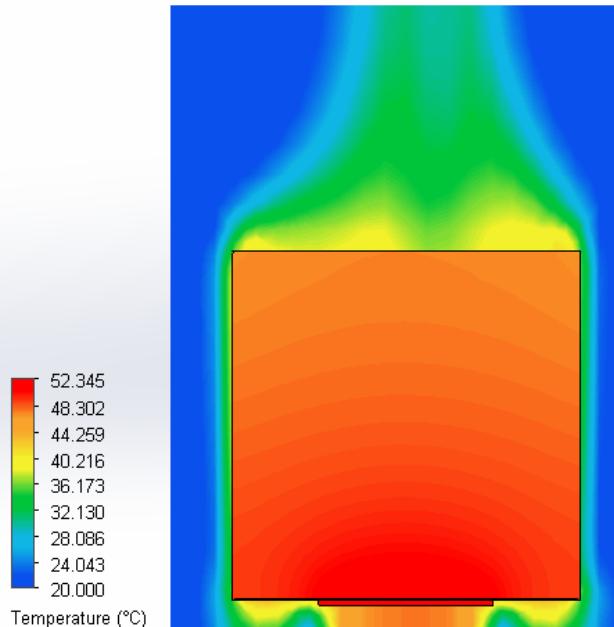
FloTHERM Cylinder  
 $T_{\text{surface}}=62.5^\circ\text{C}$



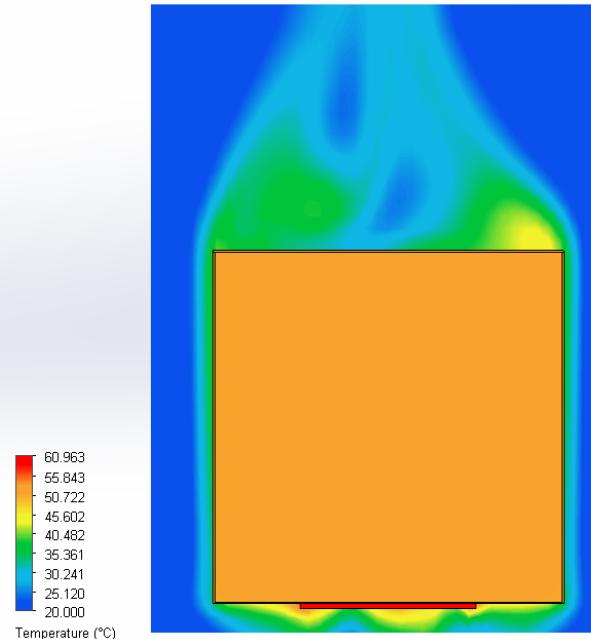
FloTHERM Cuboid  
 $T_{\text{surface}}=50^\circ\text{C}$

# Modeling Issue

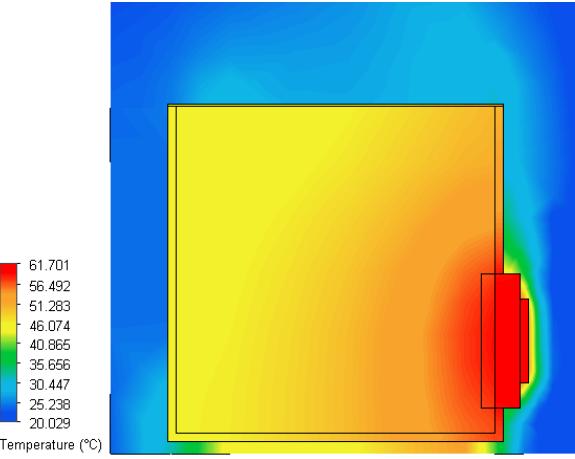
- ☞ Usually we use solid component instead of fluid to simulate the conduction
  - If the fluid stirred?
  - How to set K value for interface?



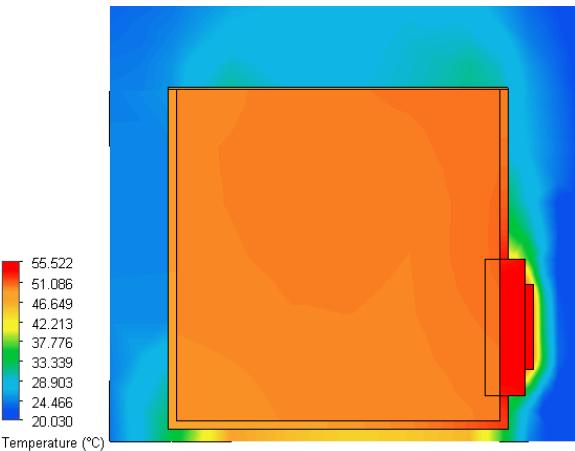
Solid Component  
 $K=0.607\text{W/mK}$



Fluid Subdomain  
Water

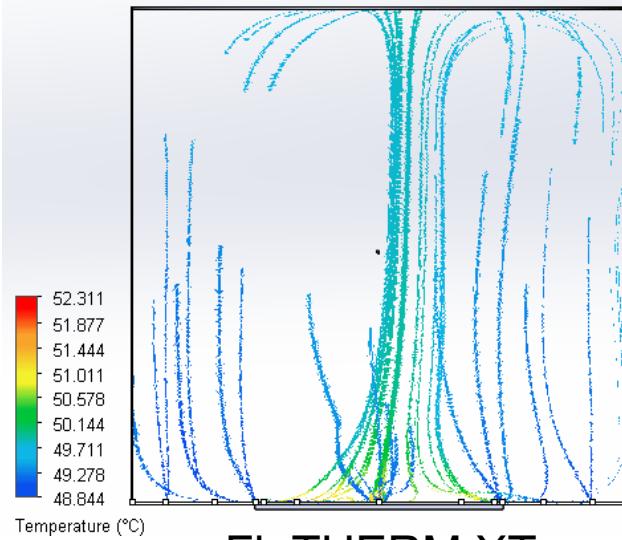


Solid Component  
 $K=0.607\text{W/mK}$



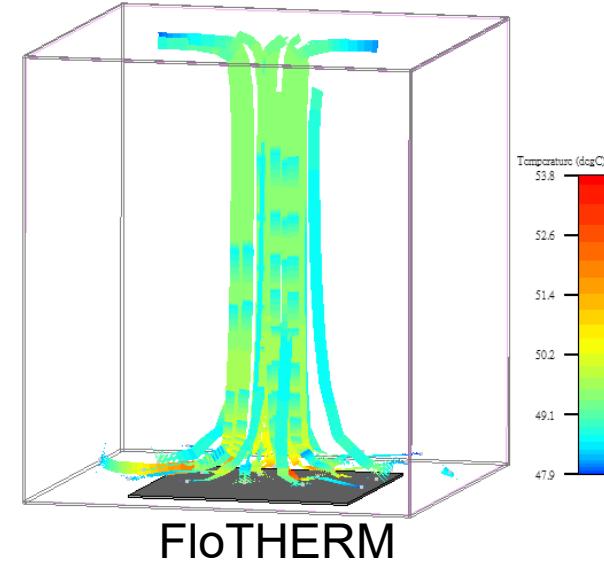
Fluid Subdomain  
Water

# Fluid Behavior- Convection



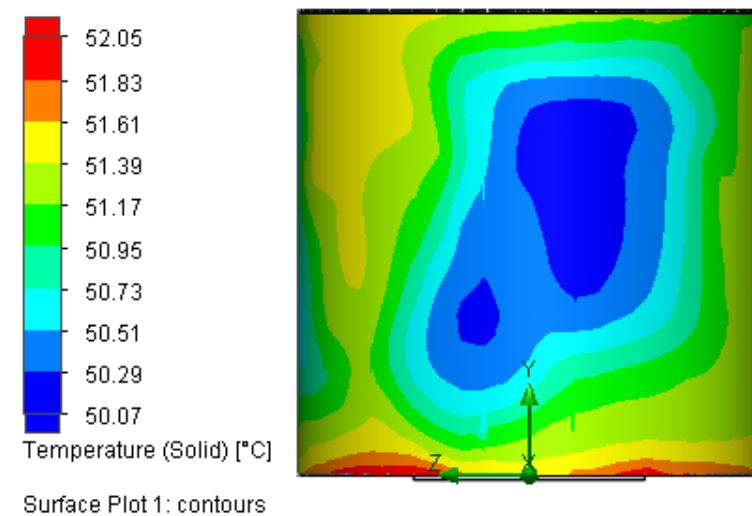
Flotherm XT (Fluid Subdomain)

- Fluid density defined by table
- Density= $f(\text{temp})$

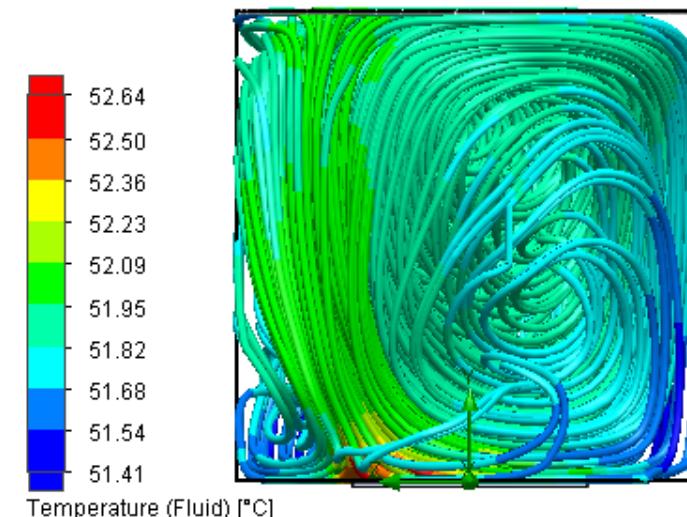


Flotherm (Region)

- Fluid density is constant
- Defined expansivity= $f(\text{temp})$

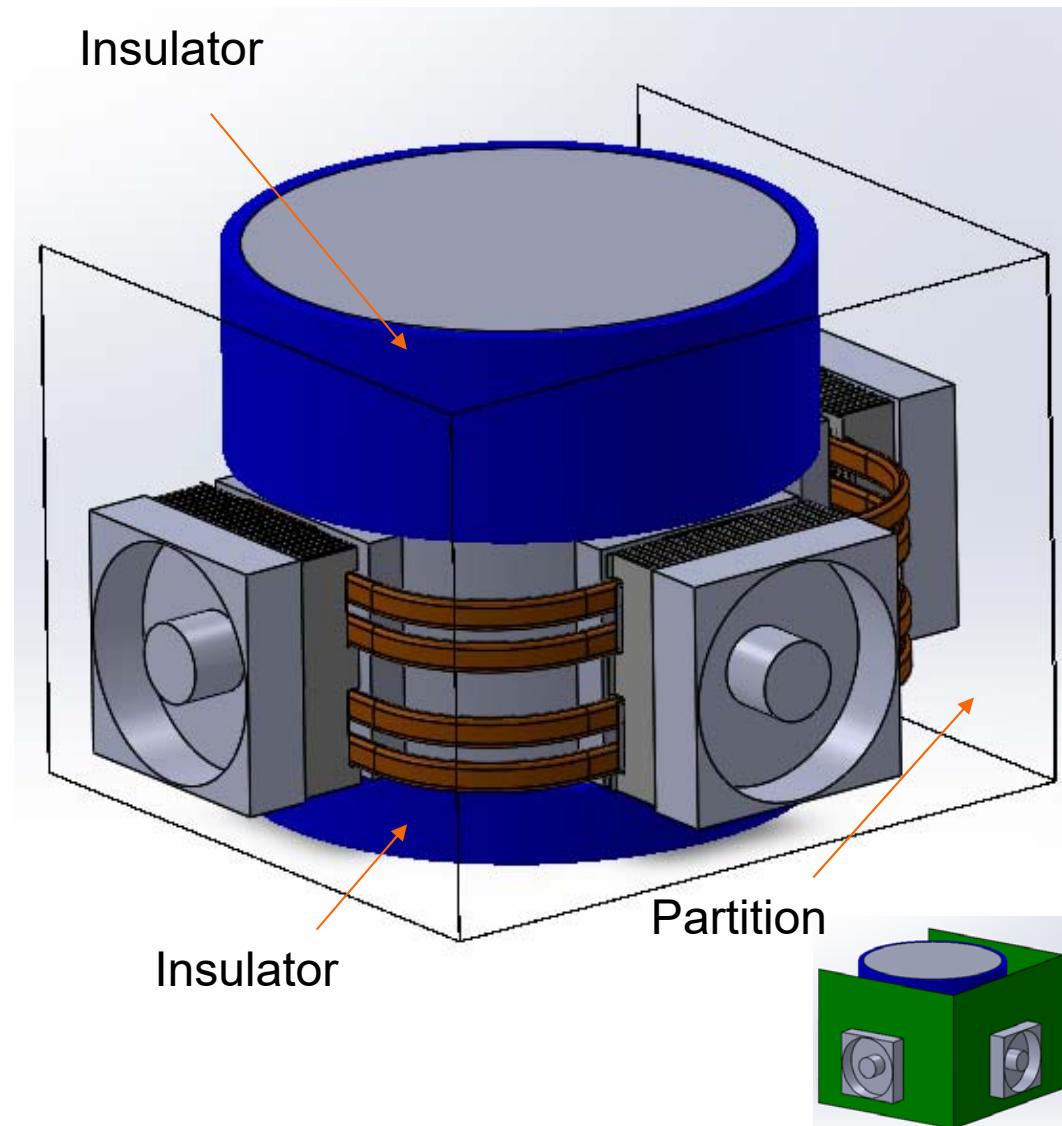
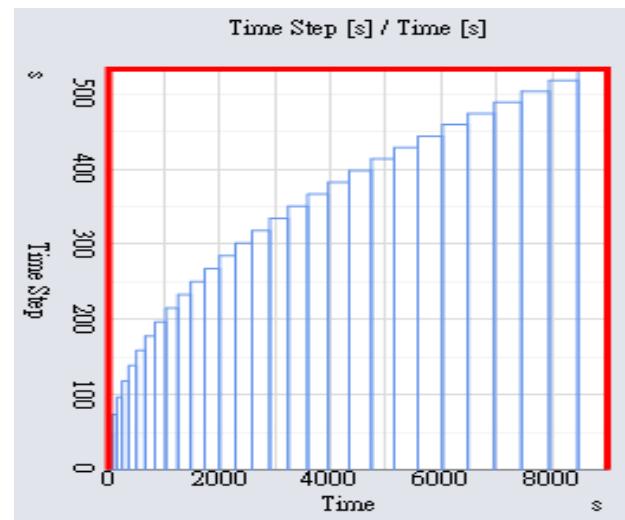


FloEFD



# System Model Setup

- ❖ Operating temperature 25°C
- ❖ 2L water,  $T_0=25^\circ\text{C}$
- ❖ 6 TECs, 3 fans, 4 heat pipes
- ❖ Neglect radiation due to the tank surface emissivity is low
- ❖ Transient simulation
  - From 0s to 9000s
  -



# TEC Setting by XT



## 🔗 TEC1-12706 Datasheet

Hot Side Temperature (°C)	25° C	50° C
Qmax (Watts)	50	57
Delta T <sub>max</sub> (°C)	66	75
I <sub>max</sub> (Amps)	6.4	6.4
V <sub>max</sub> (Volts)	14.4	16.4
Module Resistance (Ohms)	1.98	2.30

Thermo Electric Cooler

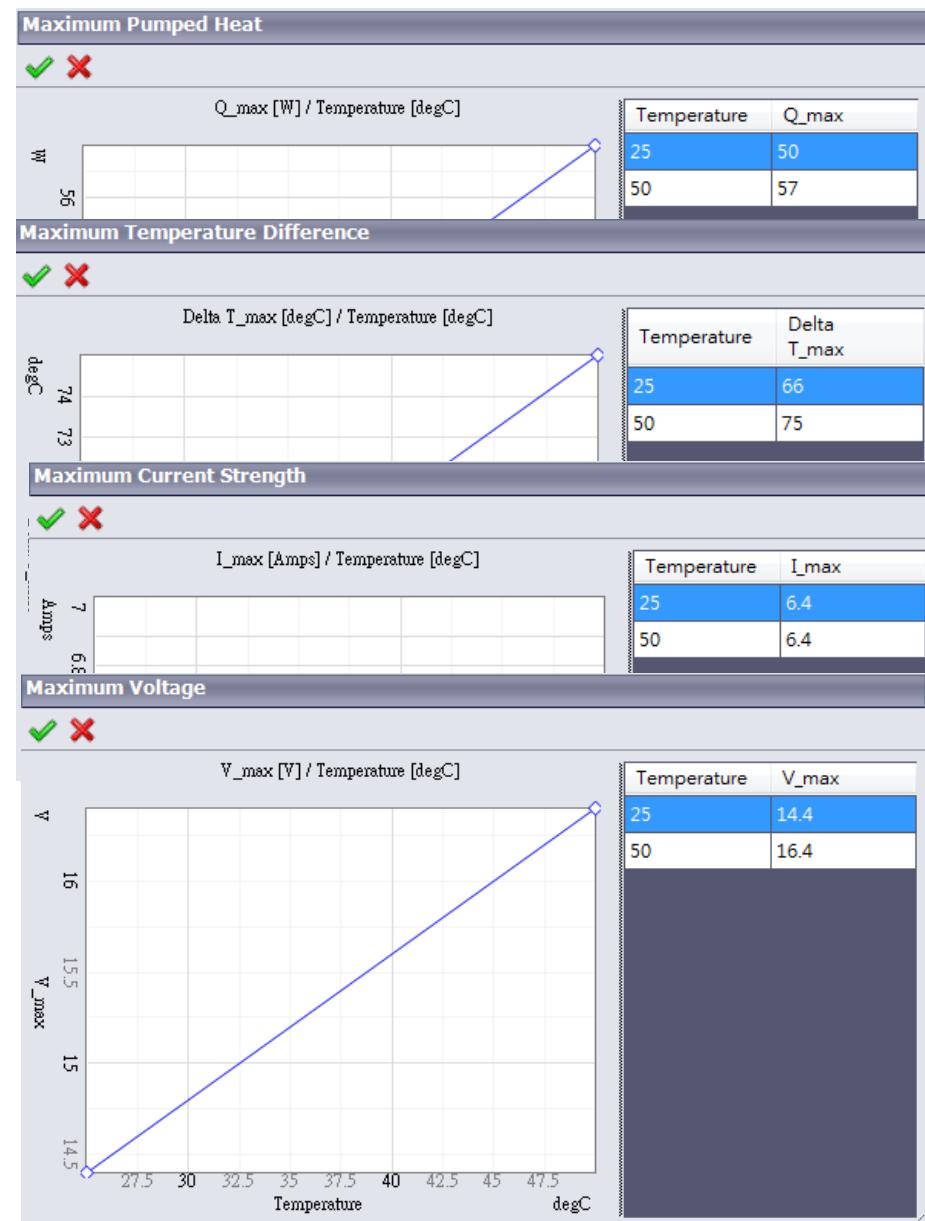
X: 40.000 mm  
Y: 40.000 mm  
Z: 3.8000 mm

Show Component Orientation

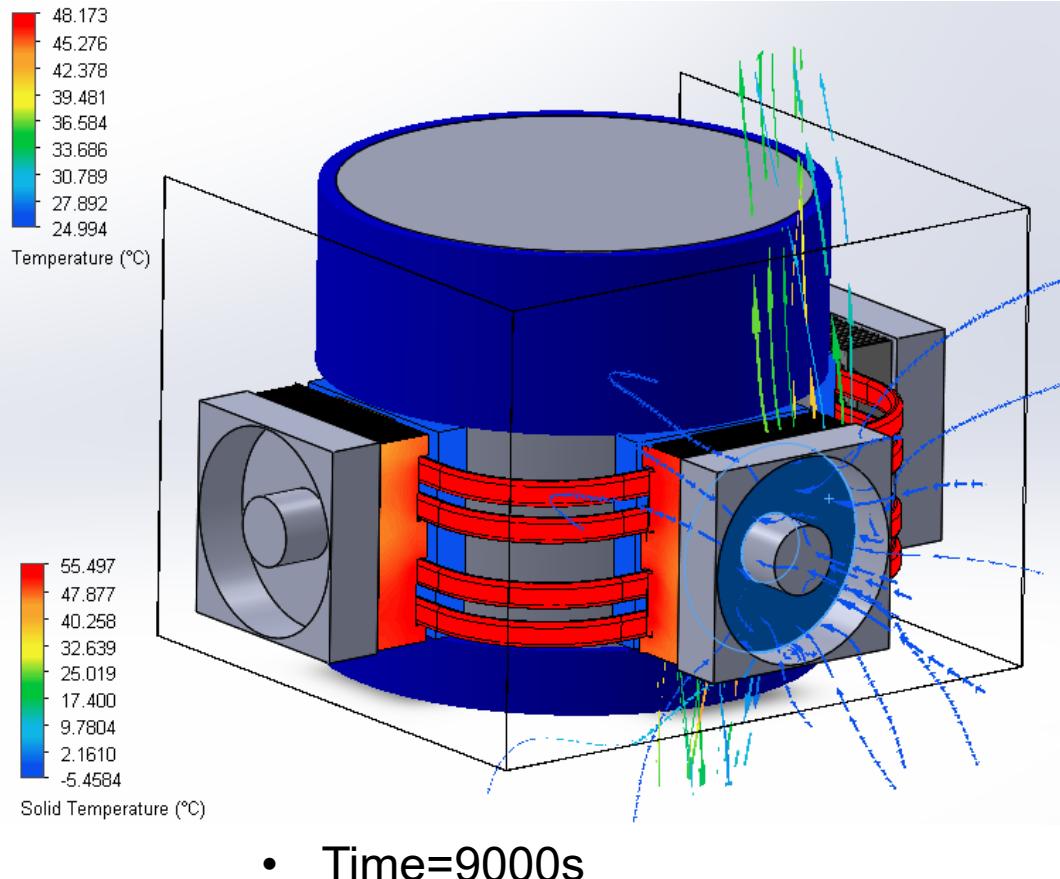
Parameters

TEC Operational Current: 6.4000 Amps

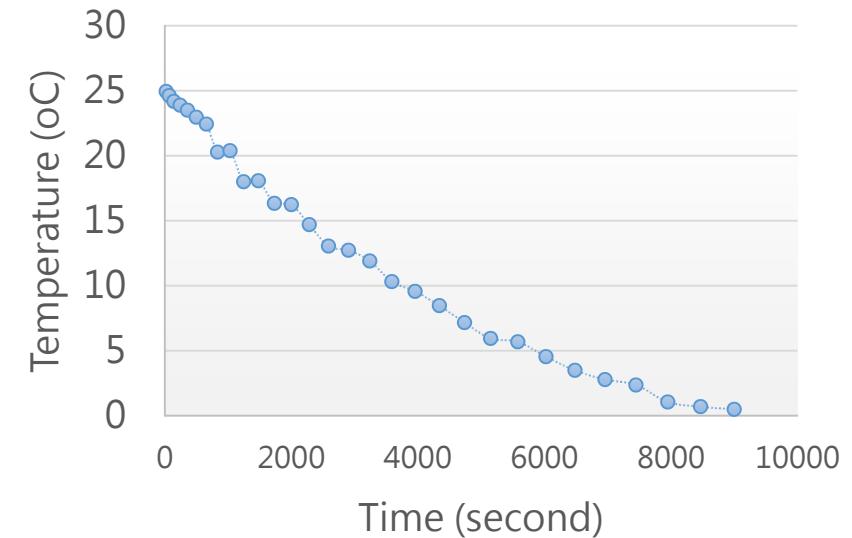
Maximum Pumped Heat  
Maximum Temperature Difference  
Maximum Current Strength  
Maximum Voltage



# Temperature Profile

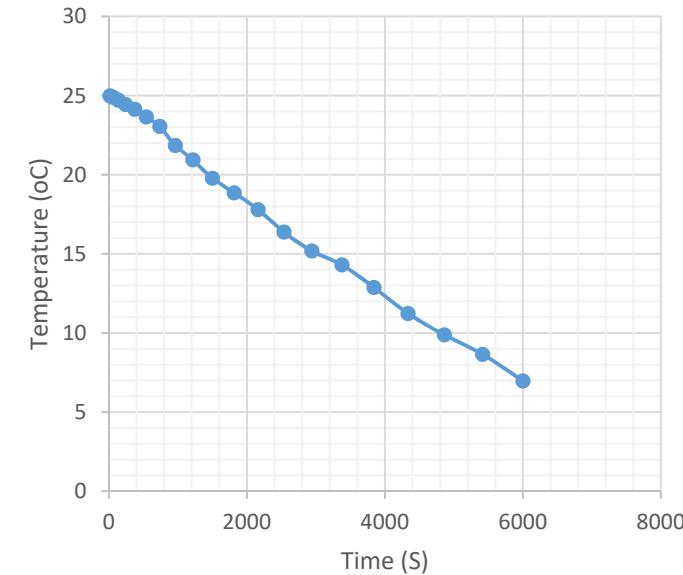
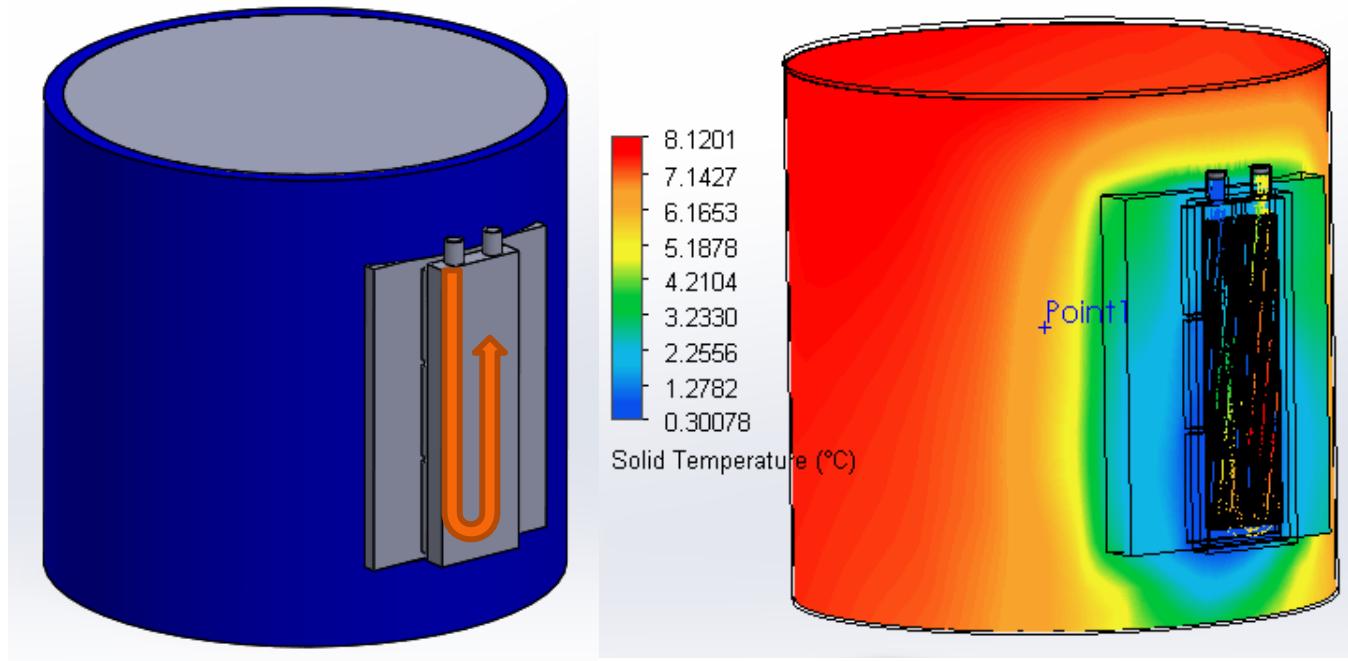


Water Center Temperature



- Still need 6000s to cool down to 4°C
- Preheated air impact the performance

# Water Cooling Solution



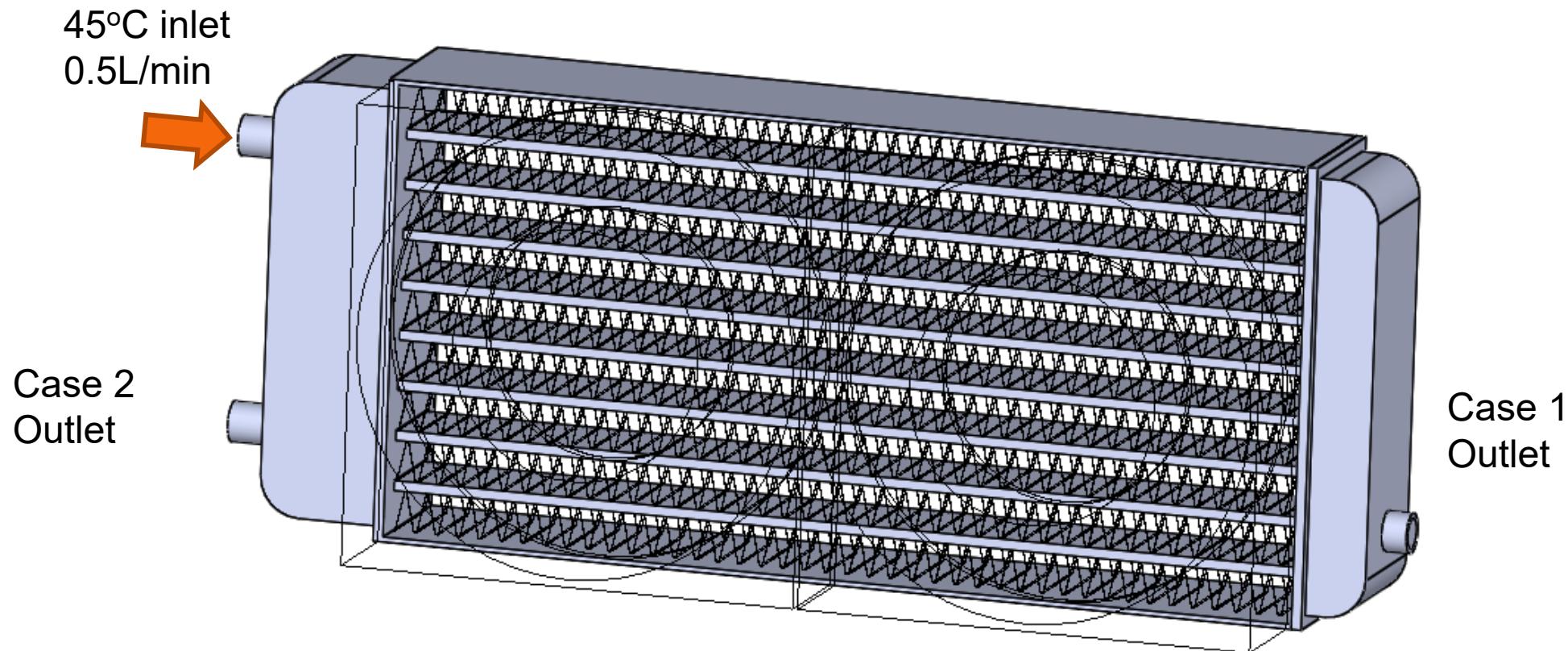
- Inlet Condition:  
Fixed flow: 1L/min, T=30°C
- Outlet Condition:  
Pressure: default ambient  
Calculate temperature 35.2°C

TEC Hot side: 39.5°C, Cold side 1.33°C  
Overall heat transfer 118W/pc

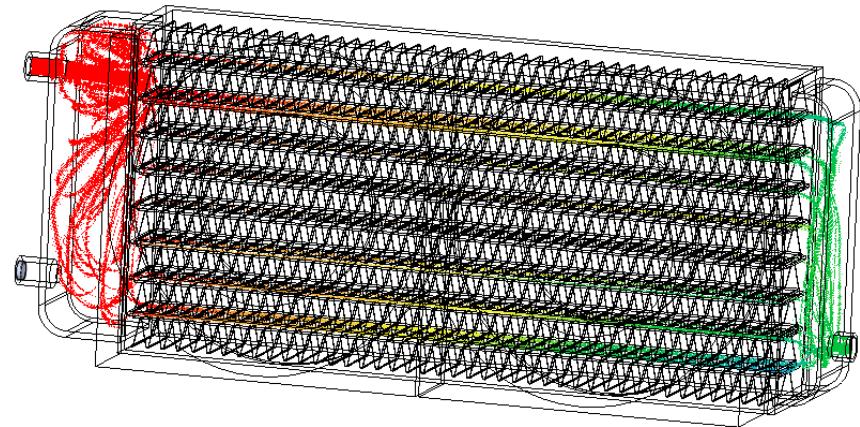
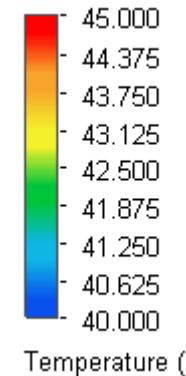
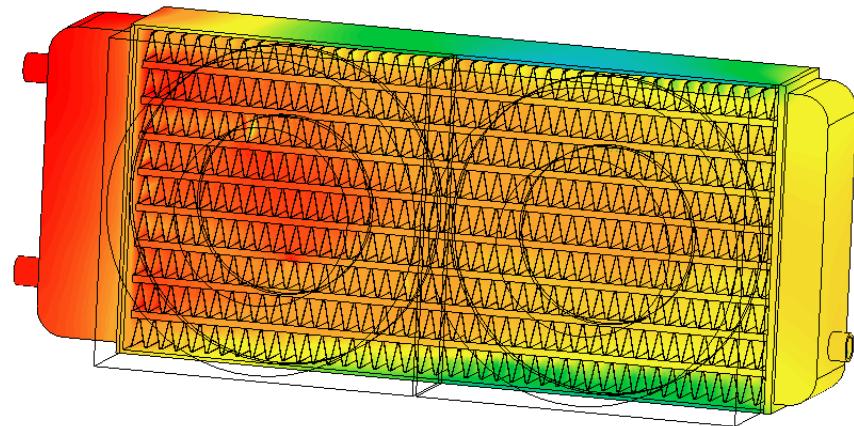
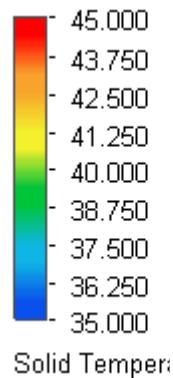
# Radiator Model



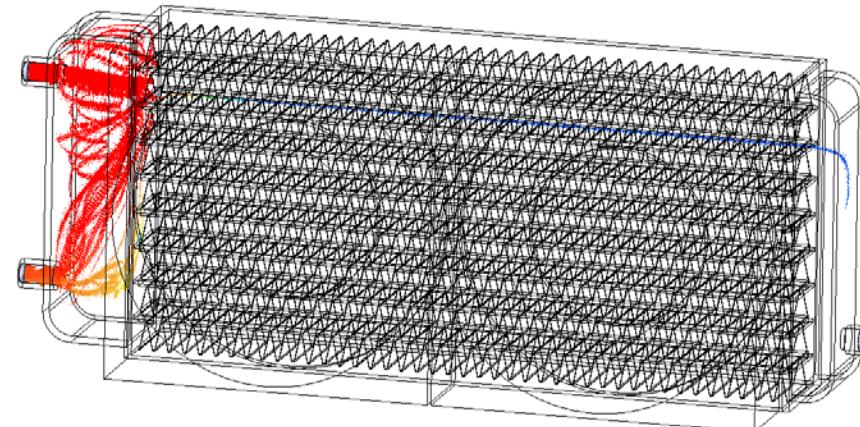
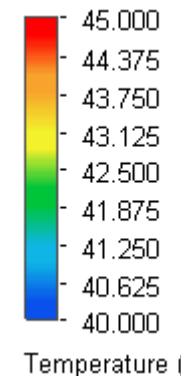
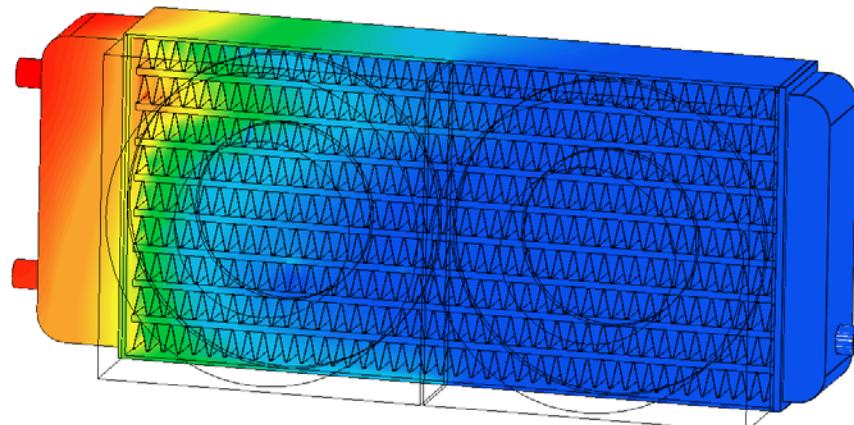
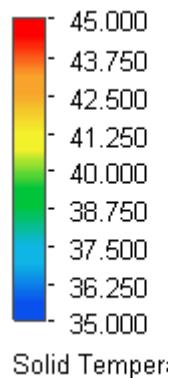
- ∞ Compare flow channel design
- ∞ 9225 fan \* 2, 35°C Ta



# Radiator Design

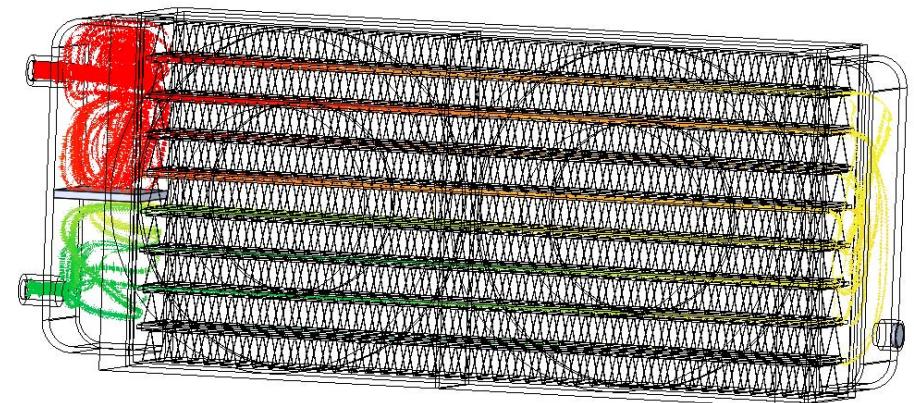
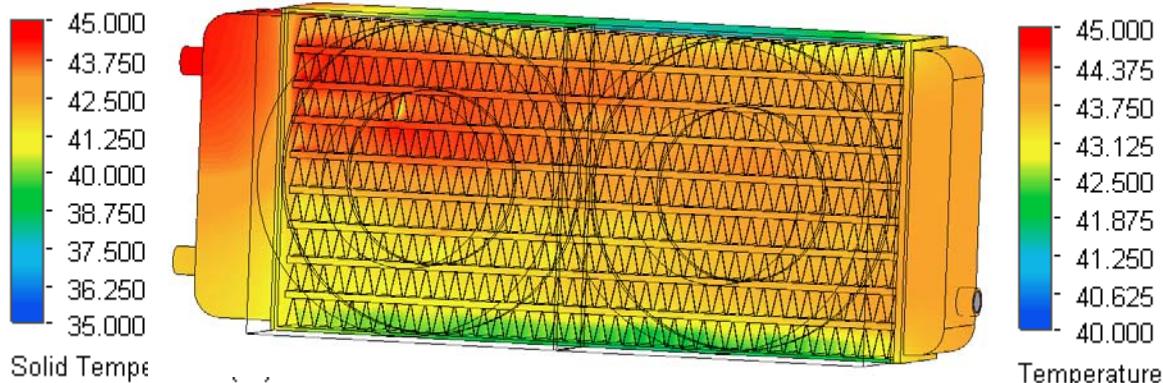


- Case 1



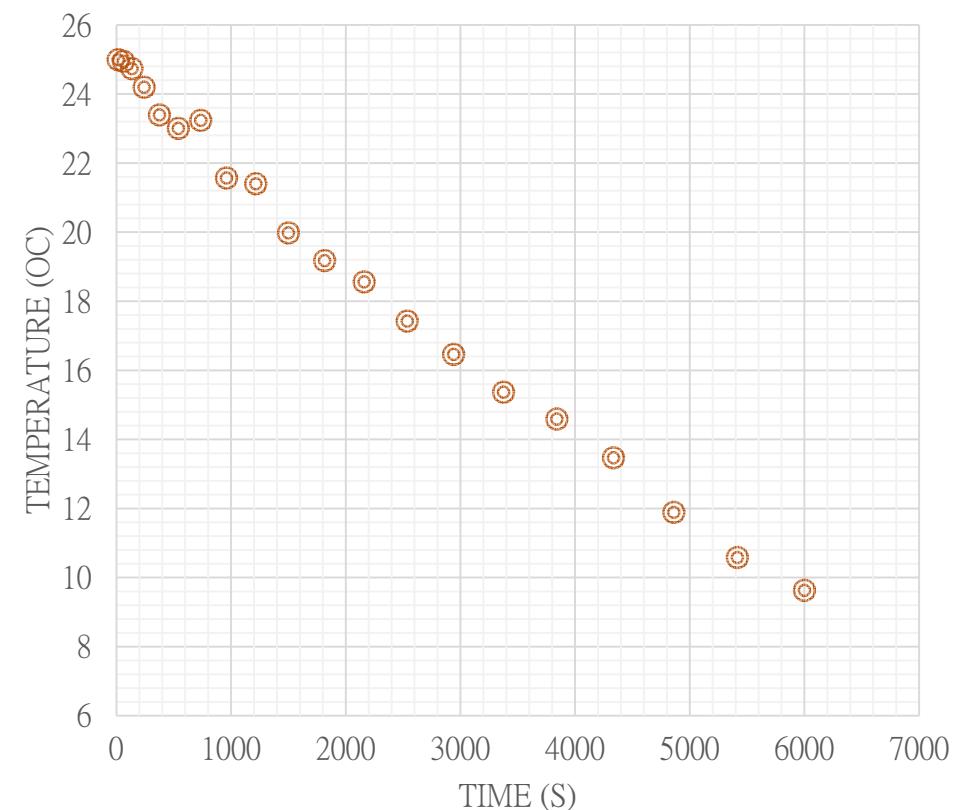
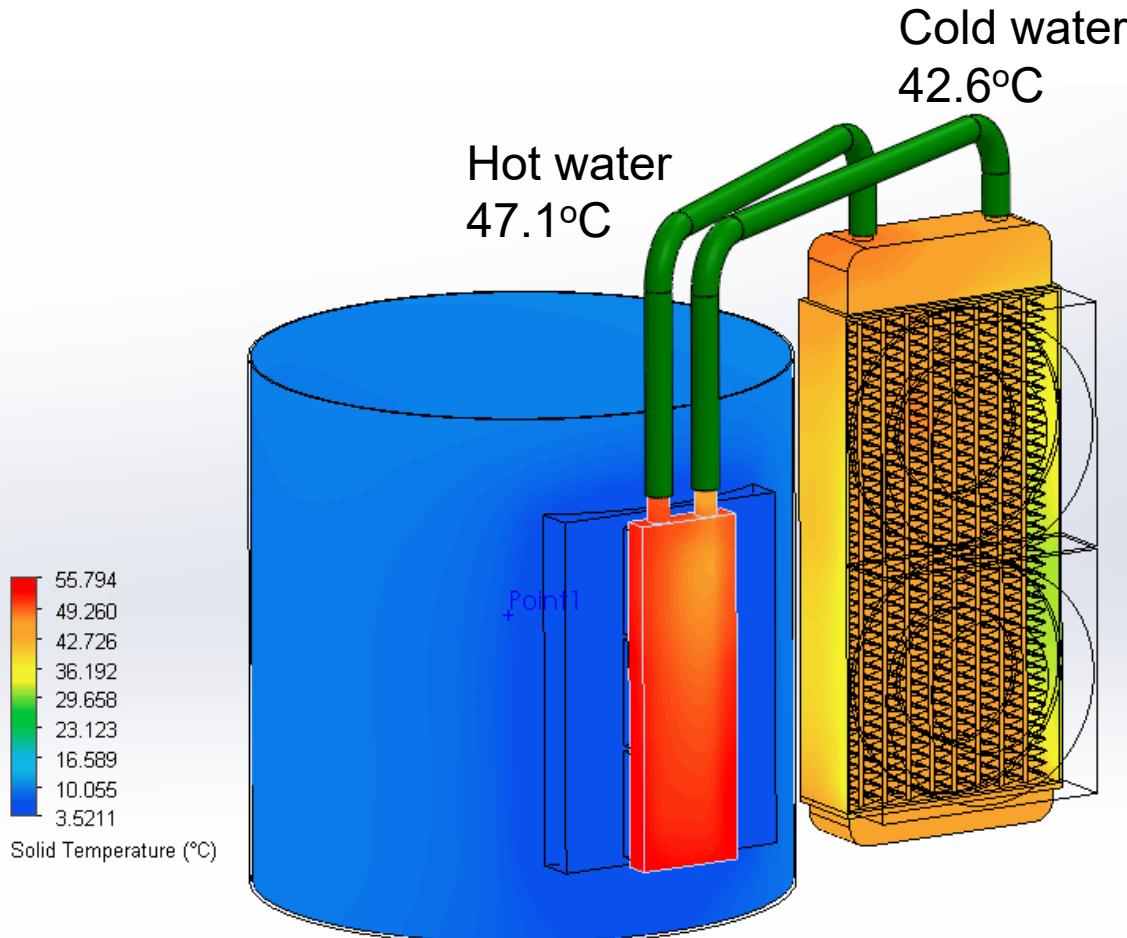
- Case 2

# Case 2 Modified



	Case 1	Case 2	Case 2
Inlet Temperature	45	45	45
Outlet	41.943	44.41	41.968
Pressure Drop (Pa)	752	761	1025

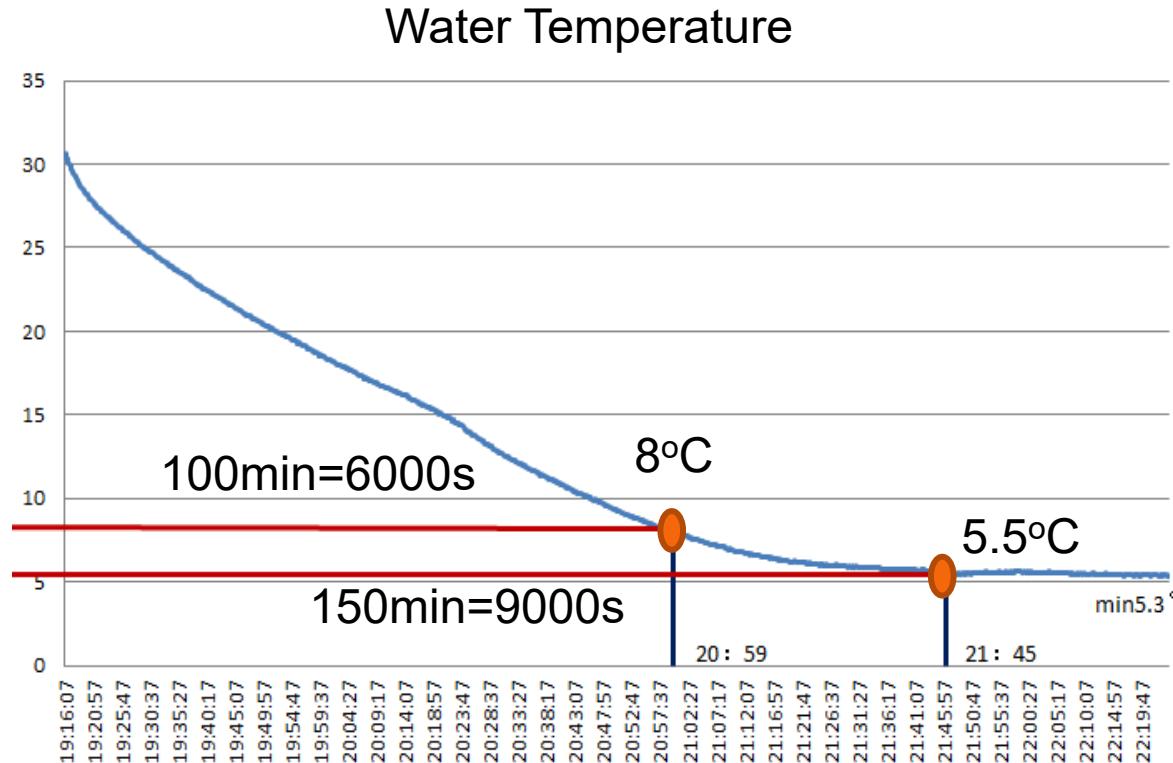
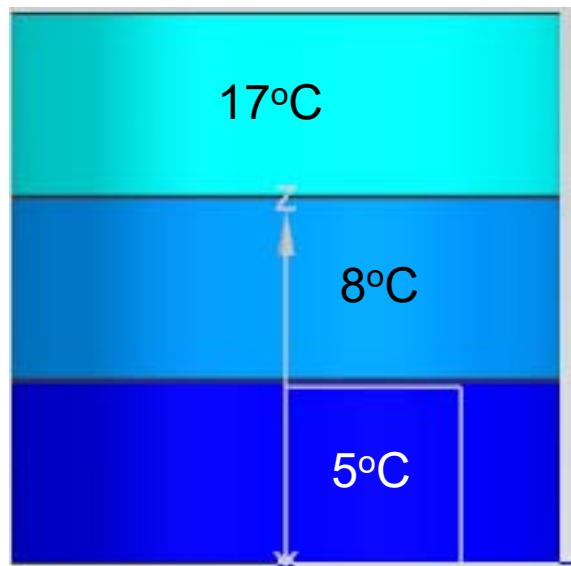
# Whole Liquid Cooling System



# Actual Test Result

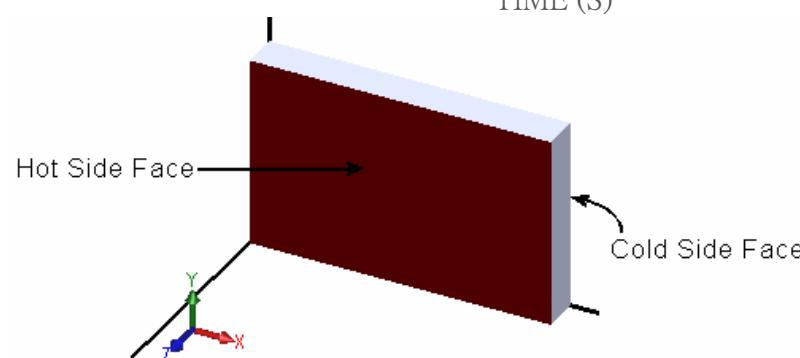
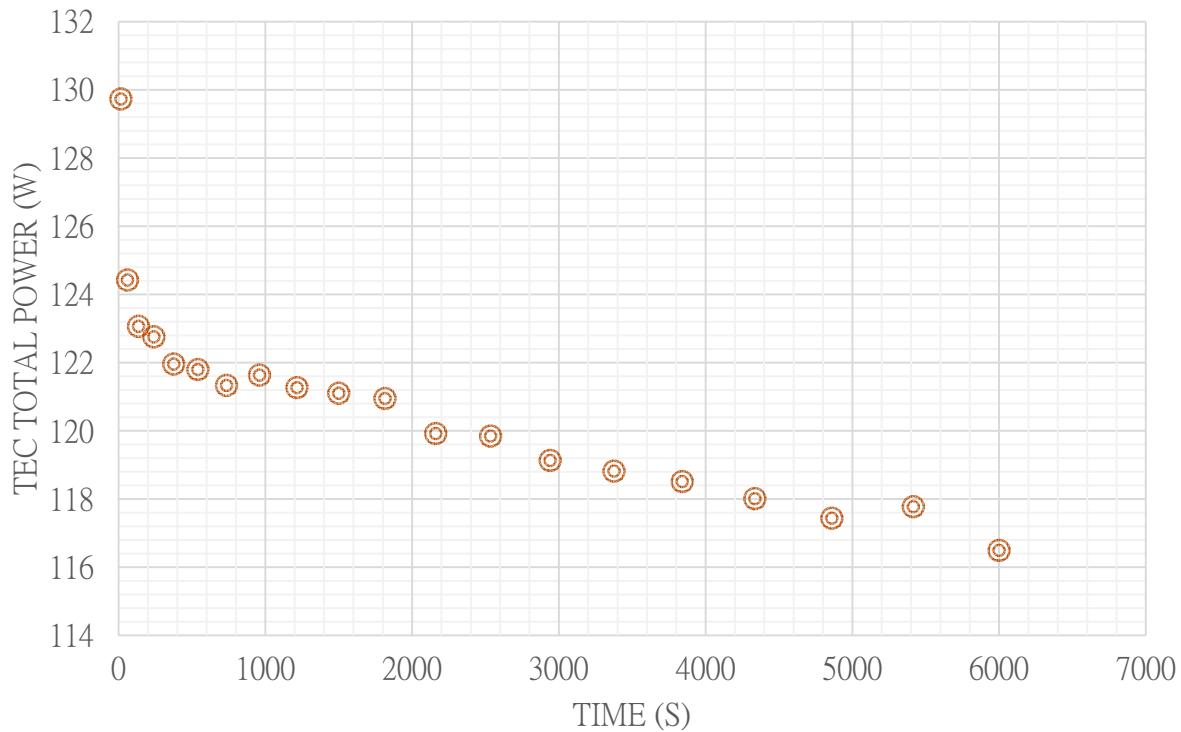


6 TECs

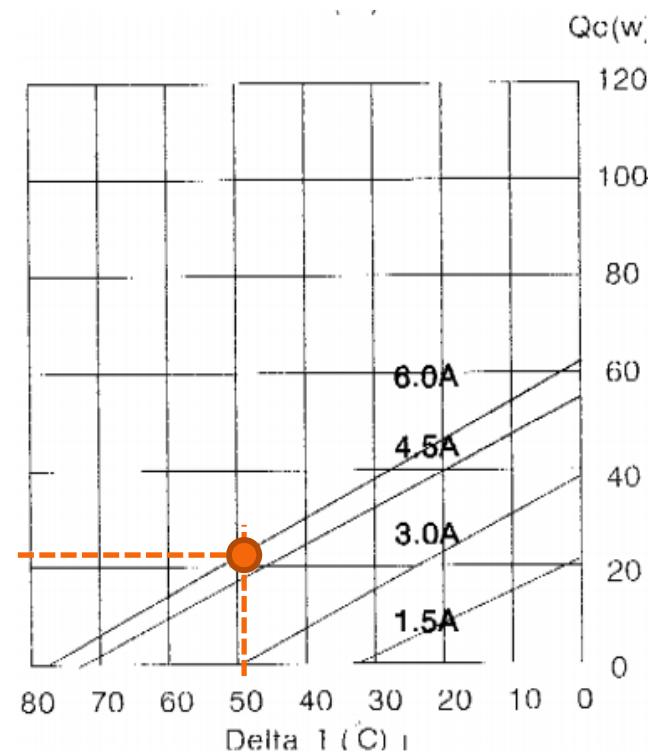


- Error analysis
  1. Without effective insulator
  2. TEC placement, constrained by system mechanical design

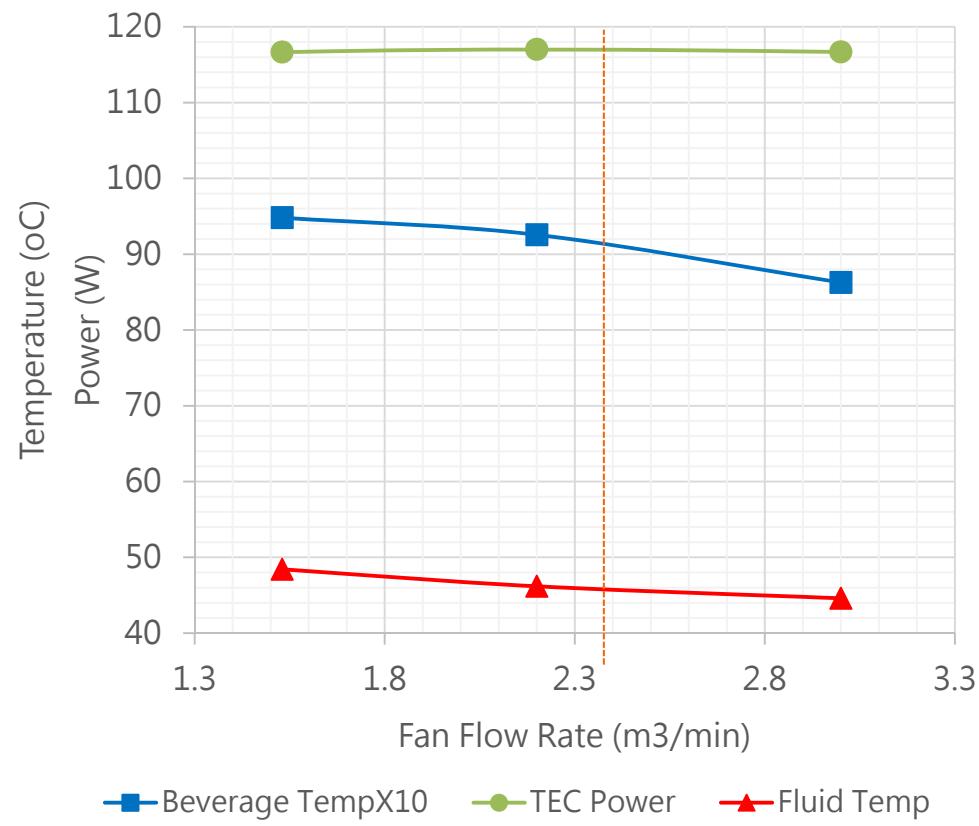
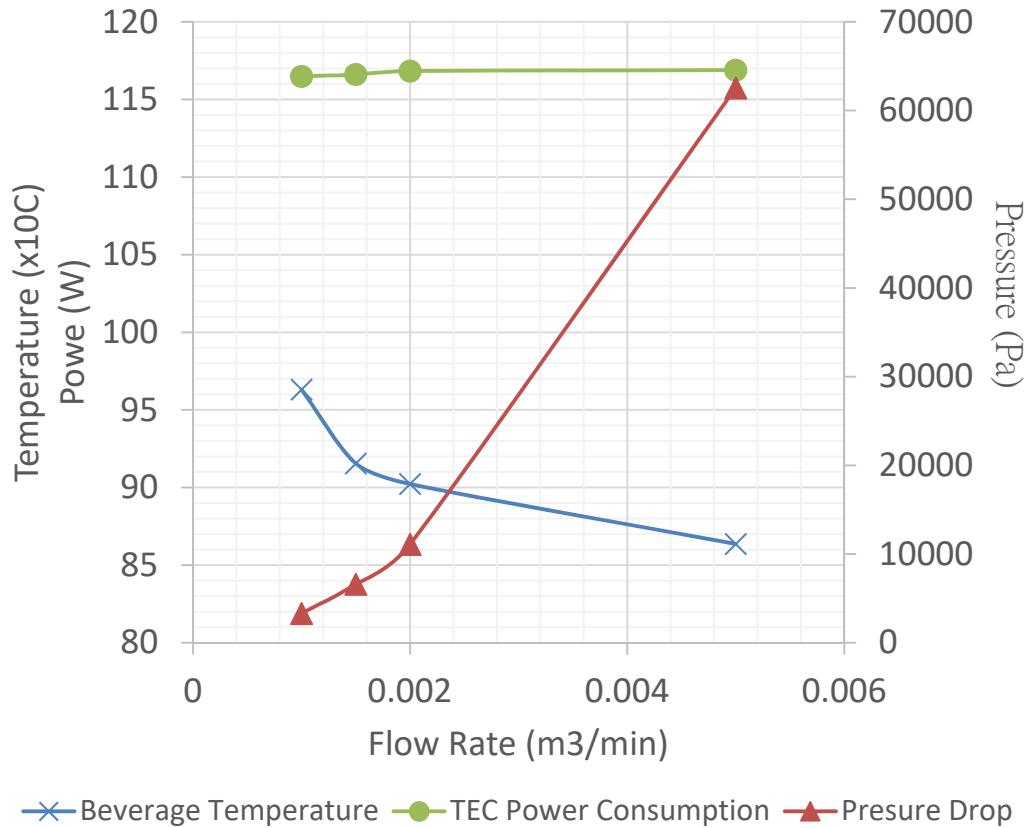
# TEC Power vs Time



- Hot side  $T=52.85^{\circ}\text{C}$
- Cold side  $T=4.014^{\circ}\text{C}$
- Heat Pumped=20.7W
- Total heat transfer rate is 117.3W



# Parametric Study

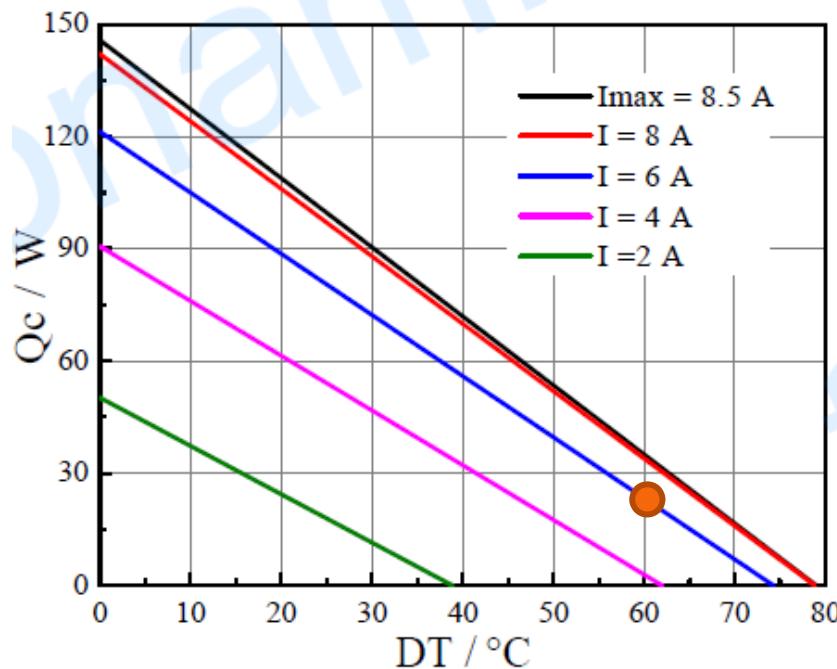


# TEC1-19908

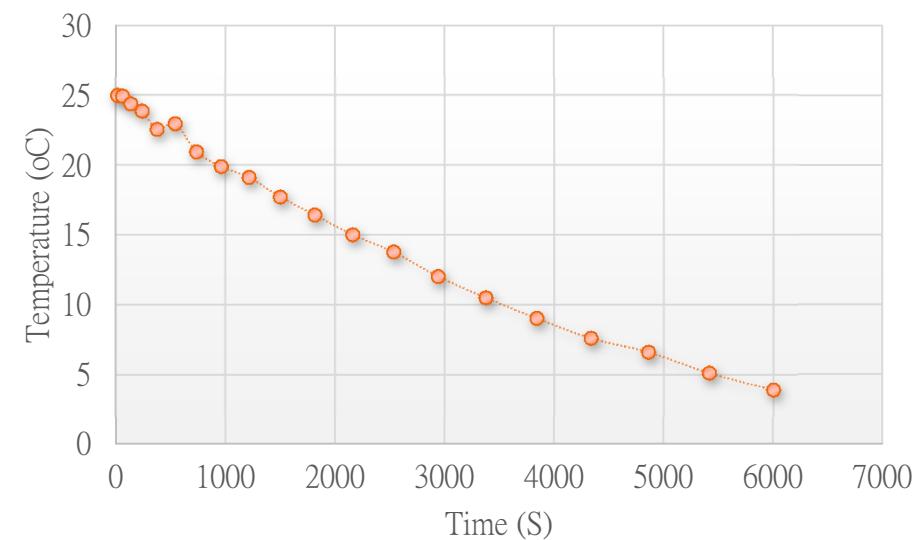


Hot side temperature (°C)	27	50
$Q_{\max}$ (Watts)	133.4	145.8
Delta T <sub>max</sub> (°C)	70	79
I <sub>max</sub> (Amps)	8.5	8.5
V <sub>max</sub> (Volts)	25	26.9
Module Resistance (Ohms)	2.25	2.49

## Performance Curves at Th=50 °C



Goals	Data Inspector	Messages	Parametric Study	Parameter	Min	Max	Average
Face	Component			Temperature (Solid) [°C]	90.118	90.118	90.118
HotFace	Thermo Electric Cooler-1			Convective/Conductive Heat Transfer Rate [W]			248.98
HotFace	Thermo Electric Cooler-1			Total Heat Transfer Rate [W]			248.98
Face < 1 >	Thermo Electric Cooler-1			Temperature (Solid) [°C]	3.4058	3.4058	3.4058
Face < 1 >	Thermo Electric Cooler-1			Convective/Conductive Heat Transfer Rate [W]			-14.841
Face < 1 >	Thermo Electric Cooler-1			Total Heat Transfer Rate [W]			-14.841
Face	Component	Parameter		Min	Max	Average	
Face < 1 >	Thermo Electric Cooler-1	Temperature (Solid) [°C]		-4.8216	-4.8216	-4.8216	
Face < 1 >	Thermo Electric Cooler-1	Convective/Conductive Heat Transfer Rate [W]				-22.974	
Face < 1 >	Thermo Electric Cooler-1	Total Heat Transfer Rate [W]				-22.974	
HotFace	Thermo Electric Cooler-1	Temperature (Solid) [°C]		57.809	57.809	57.809	
HotFace	Thermo Electric Cooler-1	Convective/Conductive Heat Transfer Rate [W]				139.73	
HotFace	Thermo Electric Cooler-1	Total Heat Transfer Rate [W]				139.73	



# Summary



- ☞ XT can perform excellent of
  - TEC behavior
  - Heat pipe
  - Fluid subdomain help to simulate the fluid convection
  - Transient model setting is easy
  - Parametric study
  - Liquid cooling combine air cooling

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